

An error in Timoshenko's "Theory of Plates and Shells"

Angus Ramsay MEng, PhD, CEng, FIMechE, Owner, Ramsay Maunder Associates; Independent Technical Editor, NAFEMS Benchmark Challenge

Edward Maunder MA, DIC, PhD, CEng, FIMechE, Consultant, Ramsay Maunder Associates; Honorary Fellow, University of Exeter, UK

Introduction

The authors recently conducted a study into the elastic behaviour of thin (Kirchhoff) plates using commercial finite-element (FE) software. In attempting to verify the FE solution, it was compared to results presented in Timoshenko's *Theory of Plates and Shells*¹ and a significant difference was observed. This article presents the work conducted to uncover the reason for this difference and reveals an error (probably typographical) in the text. The source of the error is identified and it is demonstrated how such errors might propagate into other texts on the subject of plates. The significance of the error to the practising engineer is also discussed.

Plate configuration considered

The plate considered is rectangular with an aspect ratio b/a . It is simply supported on two opposite sides and loaded with a uniformly distributed load (UDL), as shown in Figure 1.

This problem is considered in Article 48 (p. 214) of Timoshenko's text¹ and the deflections and moments at points A and B are reported in the text (Table 47, p. 219) for a Poisson's ratio of $\nu = 0.3$. This table has been reproduced in Table 1 where D is the flexural rigidity of the plate and w is the transverse displacement.

In Figure 2, an infinitesimal region around the centre of the plate is shown, together with the moments and stresses. The UDL causes sagging moments in both longitudinal and transverse directions, which induce stresses in the plate, linearly distributed across the thickness, as shown, with the stresses on the top surface both being compressive. Note that the moment m_x causes a direct stress in the x direction σ_x .

FE analysis of plate

The aspect ratio of the plate considered was 0.5 and the authors chose to study the convergence of the moment ratio (defined

in Fig. 2) with both mesh refinement and span-to-thickness ratio a/t . The reason for considering convergence with span-to-thickness ratio was that the FE system used only provided thick (Reissner-Mindlin) plates, and it was therefore necessary to ensure that the chosen thickness was small enough to have removed the influence of shear deformation, which is not considered in the thin (Kirchhoff) formulation being investigated.

The results from this convergence study are summarised in Figure 3, where an initial mesh of $1 \times 2 = 2$ elements was used with uniform mesh refinement and span-to-thickness ratios between 2 and 2000 were studied. Both studies converge to a moment ratio of 10.19 as shown in Fig. 3.

The ratio of the moments presented in Timoshenko's is 12.11, so there is a significant difference, approaching 20%, between the FE values and the published moments, and this needs further investigation.

Figure 1
Plate configuration

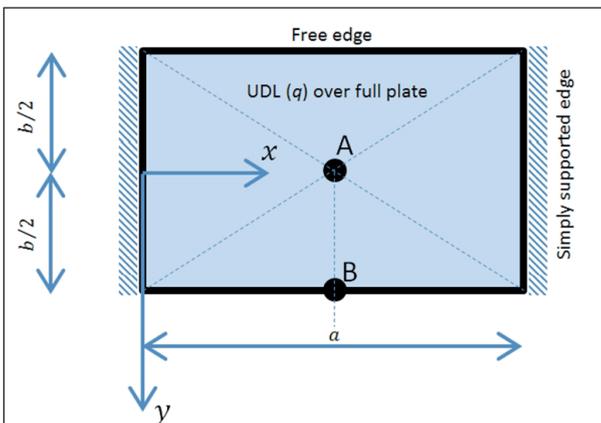


Figure 2
Moments and stresses at centre of plate (point A) and definition of moment ratio

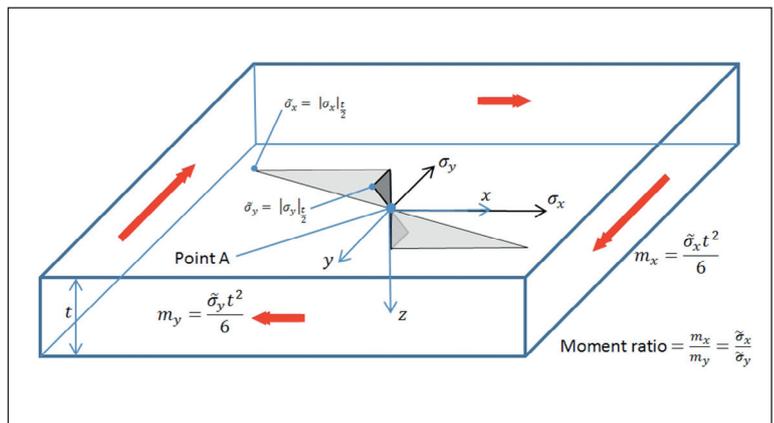


Table 1: Timoshenko's results

	Point A			Point B	
	$x = a/2; y = 0$			$x = a/2; y = \pm b/2$	
b/a	$w = \alpha_1 \frac{qa^4}{D}$	$M_x = \beta_1 qa^2$	$M_y = \beta_1' qa^2$	$w = \alpha_2 \frac{qa^4}{D}$	$M_x = \beta_2 qa^2$
	α_1	β_1	β_1'	α_2	β_2
0.5	0.01377	0.1235	0.0102	0.01443	0.1259
1.0	0.01309	0.1225	0.0271	0.01509	0.1318
2.0	0.01289	0.1235	0.0364	0.01521	0.1329
∞	0.01302	0.1250	0.0375	0.01522	0.1330

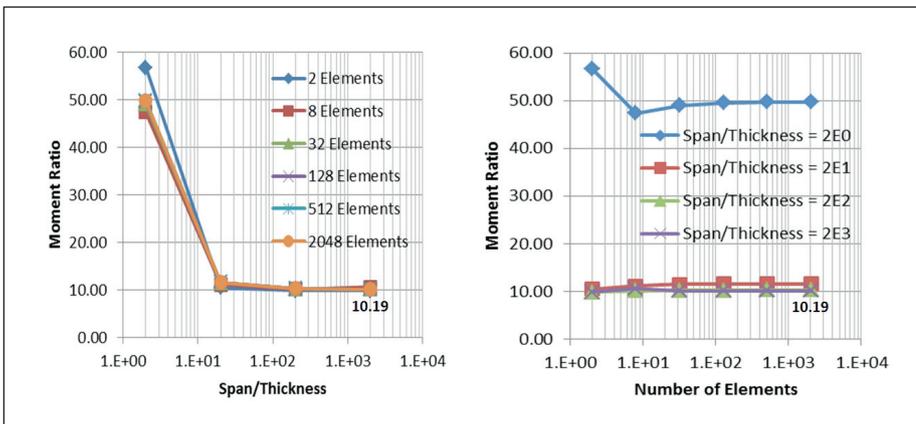


Figure 3
Convergence of moment ratio at point A (FE)

Development of computer program

Timoshenko's text provides an expression for the plate transverse displacement w as a single, infinite series (attributed to Levy and presented on p. 217, Eq.(h)), which may be twice differentiated to produce the curvatures and then, through the constitutive relations, the moments – see p. 39 for an example¹.

The expressions for the moments were coded into a small program so that they could be evaluated at a given point within a plate of arbitrary aspect ratio b/a . The summation implied by the series solution is carried out in a loop for which only odd indices are considered and the upper value of the index is maintained as a variable in the program. Rapid convergence is observed, with the moment ratio for 26 terms in the series being 10.17843 (Figure 4).

The program produces values of displacement and moment at any point. These values may be used to plot distributions across the plate, and inspection

of these distributions for satisfaction of the kinematic and static boundary conditions will provide verification that the program is correct.

The displacement field, not shown in this article for conciseness, demonstrates that the zero displacement condition along the simply supported edges is satisfied and that the field possesses the expected symmetry about the lines $x = a/2$ and $y = 0$.

The Cartesian components of moment are shown in Figure 5. The static boundary conditions require there to be zero bending

moment along all edges and this is clearly seen. The torsional moments are not required to be zero along the boundary, as Kirchhoff theory is assumed, but they should be zero along the two lines of symmetry and this is seen to be the case. The principal moments and the von Mises moment field M_{vm} are also shown in the figure and it is seen that the point of first yield is at point B, the centre of the free edges.

An additional FE result was produced using a pure Kirchhoff finite element. This gave a moment ratio of 10.1784 which, to four decimal places, is identical to that produced by the program, thus independently verifying the program. The moment ratios from the four independent sources considered are shown in Table 2.

The results shown in Table 2 indicate that there is something amiss with the values published in Timoshenko's text, at least for an aspect ratio of 0.5, and further investigation of the individual moment components used in the moment ratio show that it is the value of M_x at point A which is in error, with the value in the text being 0.0102 and the value from the program being 0.0122.

The table of point results produced in Timoshenko (and reproduced in Table 1) is attributed to a 1936 publication by Holl² studying the problem presented in

Table 2: Summary of moment ratios at point A from different sources

Source	Moment ratio at point A
Timoshenko	12.11
FE (Reissner–Mindlin)	10.19
Program	10.18
FE (Kirchhoff)	10.18

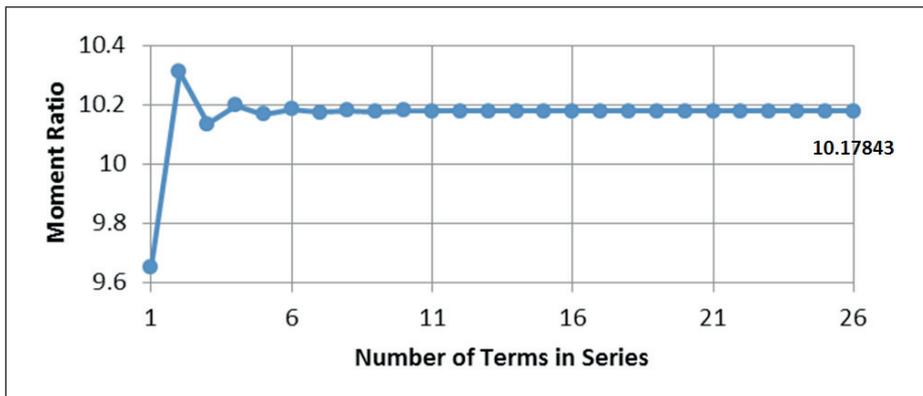


Figure 4 Convergence of moment ratio at point A (program)

in the book value, since, if the last two digits are transposed, then it becomes 0.0120 and the error reduces to -1.64%, which is much more in line with the error in the other values reported in Table 3.

Practical conclusions

This article has uncovered, by chance, an error in the published result for the transverse moment at the centre of the plate configuration considered when the aspect ratio is 0.5. It illustrates the sort of care required by practising engineers when taking published data at face value, even when it comes from such revered texts as Timoshenko's¹. With the wide availability of FE systems, the practising engineer can, and should, check the values they are going to use in the design or assessment of a structural member. It is also interesting to note the fact that published errors can propagate. In this case, erroneous data presumably first published in the first edition is still being used in the 28th reprint of Timoshenko's text¹ published in 1989 and also appears in Szilard's 2004 publication on *Theories and Applications of Plate Analysis*³ (case number 103).

The authors of this current article have contacted the publishers of Timoshenko's text¹ regarding this error, asking whether it might be corrected at a future reprint. However, it is understood that no further reprints are likely. This raises the question of how one then might protect practising engineers against the propagation of erroneous published data. One way to

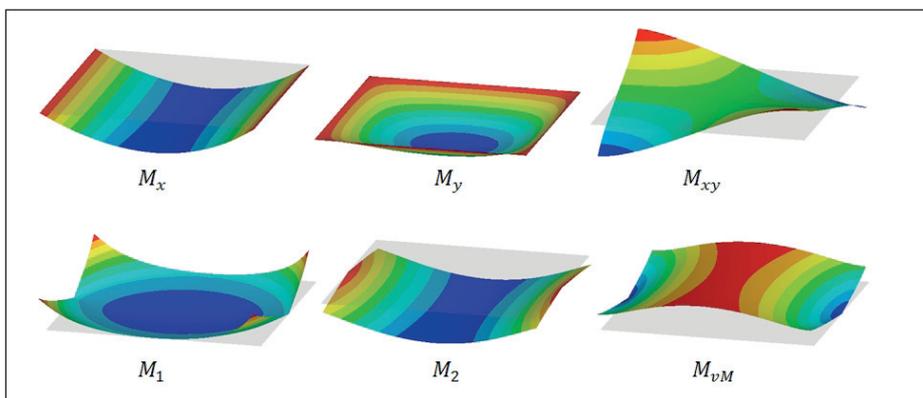


Figure 5 Moments from program

this article. In the era when the original publication was prepared, digital calculators/computers were not available and so it is likely that the moment values were calculated by hand using tabulated data to obtain the hyperbolic trigonometrical functions and taking only a small (but presumably sufficient) number of terms

in the series. The difference between the published values and those produced by the program is reasonably small for all but M_y for an aspect ratio of 0.5, as shown in Table 3.

As already noted, the values of M_y at point A for the book and program are, respectively, 0.0102 and 0.0122. It is interesting to surmise that there is a typographical error

Table 3: Percentage difference in displacements and moments

		Point A			Point B	
		$x = a/2 ; y = 0$			$x = a/2 ; y = \pm b/2$	
b/a	$w = \alpha_1 \frac{qa^4}{D}$	$M_x = \beta_1 qa^2$	$M_y = \beta'_1 qa^2$	$w = \alpha_2 \frac{qa^4}{D}$	$M_x = \beta_2 qa^2$	
	α_1	β_1	β'_1	α_2	β_2	
0.5	0.41	-0.12	-16.03	-1.47	-1.50	
1.0	-0.03	-0.04	0.08	0.52	0.54	
2.0	0.02	0.03	0.03	0.05	0.07	
∞	-0.01	-0.05	-0.05	0.01	-0.01	

Notes
 (i) Percentage differences calculated as $100 \times (\text{book} - \text{program})/\text{program}$
 (ii) The values in the table converge very rapidly with increasing aspect ratio and the value used for the "infinite" aspect ratio was 10.

do this would be to have an online repository of such errors which engineers can access to check that there are no reported errors in the data they are proposing to use. In the absence of such a facility, the best one can do is publish the finding, as here, with the hope that it will reach the intended audience.

With regard to the engineering significance of this finding, the error leads to an under-prediction of the minor (transverse) component of the moment at the plate centre. The engineer designing a steel plate might use the moments to calculate the von Mises moment and ensure that this is below the yield moment for the steel being used. Since the von Mises moment is greater at the centre of the free edge (point B) than at the centre of the plate, then provided the engineer notices this, the erroneous value in the table would never be used. For a designer of a reinforced-concrete slab, however, this moment value may well be used to size the reinforcement lying parallel to the y axis and an under-

prediction of some 16% might lead to a situation where the structure is pushed out of the elastic region and into the plastic region. The degree to which this will occur should, however, be well within the ultimate capacity of the slab, but may be undesirable in terms of serviceability issues such as cracking of the concrete.

Acknowledgements

The authors would like to thank:

Professor J.P.B.M. Almeida of the Department of Civil Engineering and Architecture, IST, University of Lisbon,

Portugal for running the plate through his Kirchhoff equilibrium FE software.

Professor Husain Jubran Al-Gahtani of the King Fahd University of Petroleum and Minerals, Dhahran, in the Kingdom of Saudi Arabia for independently verifying the results produced by the program using Mathematica.

Contact details

Angus Ramsay:

angus_ramsay@ramsay-maunders.co.uk

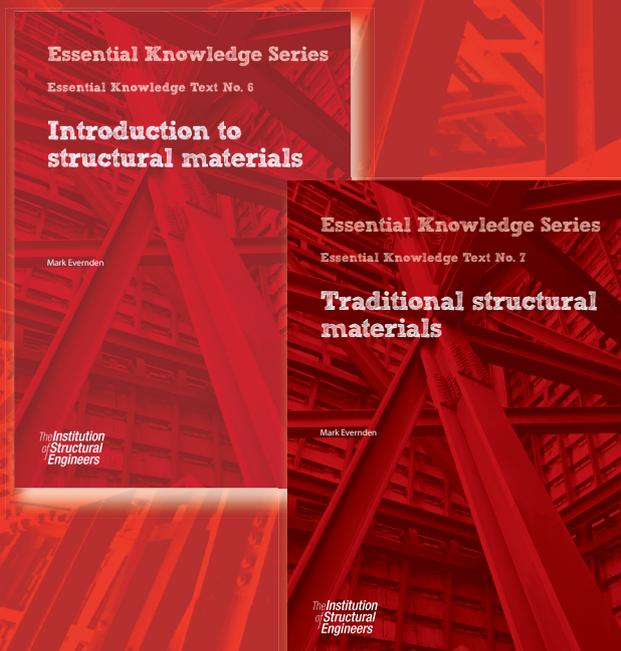
Edward Maunder:

e.a.w.maunder@exeter.ac.uk

References

- ▶ 1 Timoshenko S.P. and Woinowsky-Krieger S. (1989) *Theory of Plates and Shells* (2nd ed.), New York, USA: McGraw-Hill
- ▶ 2 Holl D.L. (1936) 'Analysis of thin rectangular plates supported on opposite edges', *Iowa Engineering Experiment Station Bulletin*, 129
- ▶ 3 Szilard R. (2004) *Theories and Applications of Plate Analysis*, New Jersey, USA: Wiley

Have you discovered the Essential Knowledge Series yet?



Break down the fundamentals of structural engineering with these concise and accessible study texts.

Free to Student and Academic Members, and available to purchase for all other members and non-members.

Get your copies here:
istructe.org/Essential-Knowledge